

# Chemical Process Control

## Auxiliary material for hand calculation practices

### GENERAL NOTIONS

#### Notation

- t = time
- T = time constant
- s = complex variable

#### Processes



The process is 'known' if function  $y(t) = f[x(t)]$  is known.

Equivalent information is given by its transfer function  $G(s)$  in Laplace domain:

$$Y(s) = G(s) \cdot X(s)$$

#### Laplace transformation

It transforms any linear differential equation to an ordinary equation.

Two-sided Laplace transformation:  $F(s) = \mathcal{L}[f(t)] = \int_{-\infty}^{\infty} f(t) \cdot e^{-st} \cdot dt$

One-sided Laplace transformation:  $F(s) = \mathcal{L}[f(t)] = \int_0^{\infty} f(t) \cdot e^{-st} \cdot dt$ , this one is simpler if

$f(x)=0$  at  $t<0$ . This condition is achieved if there is (was) steady state before  $t=0$  and deviation variable are used:  $\hat{x}(t) = x(t) - \bar{x}(t)$

$$f(t) \xrightarrow{\mathcal{L}} F(s) \xrightarrow[\text{used in our calc. practices}]{\text{calculation}} G(s) \xrightarrow{\mathcal{L}^{-1}} g(t)$$

#### Linearity:

$$a \cdot f(t) + b \cdot g(t) \xrightarrow{\mathcal{L}} a \cdot F(s) + b \cdot G(s)$$

#### Useful particular cases:

$$y(t) \xrightarrow{\mathcal{L}} Y(s)$$

$$\frac{dy(t)}{dt} \xrightarrow{\mathcal{L}} s \cdot Y(s)$$

$$\int y(t) dt \xrightarrow{\mathcal{L}} \frac{Y(s)}{s}$$

$$\left( \delta(t) = \frac{d1(t)}{dt} \right) \xrightarrow{\mathcal{L}} 1 \quad \text{(unit impulse)}$$

$$1(t) \xrightarrow{\mathcal{L}} \frac{1}{s} \quad \text{(unit step)}$$

$$(t = \int 1(t) dt) \xrightarrow{\mathcal{L}} \frac{1}{s^2} \quad \text{(unit slope ramp)}$$

$$e^{a \cdot t} \xrightarrow{\mathcal{L}} \frac{1}{s - a}$$

## I.-II. ORDER ELEMENTS

### Proportional element

$$y(t) = A \cdot x(t)$$

$$G(s) = \frac{Y(s)}{X(s)} = A$$

### First order element

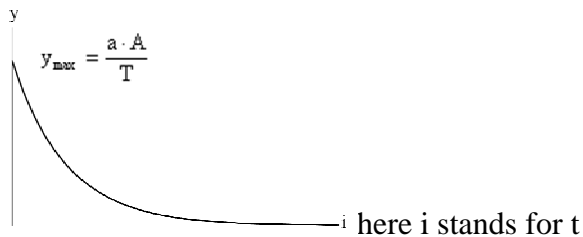
Differential equation:  $T \frac{dy(t)}{dt} + y(t) = A \cdot x(t)$

Transfer function:  $G(s) = \frac{A}{Ts + 1}$

Impulse response:  $\hat{y} = \frac{a \cdot A}{T} \cdot e^{-\frac{t}{T}}$

Step response:  $\hat{y} = a \cdot A \cdot \left[ 1 - e^{-\frac{t}{T}} \right]$

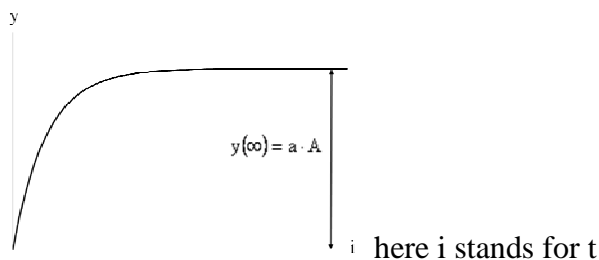
Ramp response:  $\hat{y} = a \cdot A \cdot T \cdot \left[ \frac{t}{T} + e^{-\frac{t}{T}} - 1 \right]$



Its step response:

$$Y(s) = X(s) \cdot G(s) = \frac{a}{s} \cdot \frac{A}{T \cdot s + 1}$$

$$\hat{y}(i) = a \cdot A \cdot \left[ 1 - e^{-\frac{t}{T}} \right]$$



## Identification from step response

### Gain

From how high the response signal increases (in limit)

Time constant: several ways you can do it

#### 1. Substitution

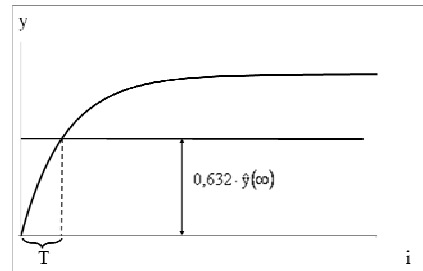
From any  $t-y(t)$  value pair (a point in the plane) you can calculate it.  
This is subject to errors.

#### 2. Ratio calculation

If  $t=T$  then

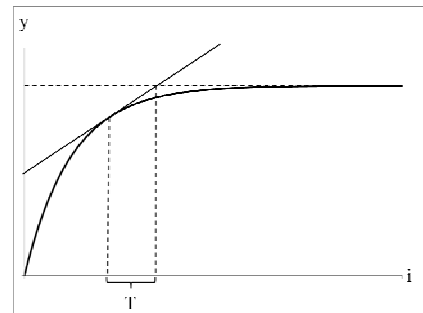
$$\hat{y}(T) = a \cdot A \cdot \left[ 1 - e^{-\frac{T}{T}} \right] = \hat{y}(\infty) \cdot [1 - e^{-1}]$$

$$\frac{\hat{y}(T)}{\hat{y}(\infty)} = 1 - e^{-1} \approx 0.632$$



#### 3. Slope fitting

(To any point)



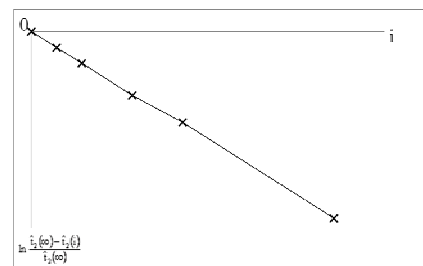
#### 4. Linearization

$$\hat{y}(t) = a \cdot A \cdot \left[ 1 - e^{-\frac{t}{T}} \right] = \hat{y}(\infty) \cdot \left[ 1 - e^{-\frac{t}{T}} \right]$$

$$\frac{\hat{y}(t)}{\hat{y}(\infty)} = 1 - e^{-\frac{t}{T}}$$

$$\frac{\hat{y}(\infty) - \hat{y}(t)}{\hat{y}(\infty)} = e^{-\frac{t}{T}}$$

$$\ln \left[ \frac{\hat{y}(\infty) - \hat{y}(t)}{\hat{y}(\infty)} \right] = -\frac{1}{T} \cdot t$$



The process is of first order if the points fall to a straight line, otherwise its behaviour is different.

Slope of the straight line:  $-\frac{1}{T}$ .

This method is best for cancelling measurement errors.

### Examples

jacketed vessel, mixed vessel, CSTR, thermometer

## Second order element

Differential equations:

$$T_2^2 \frac{dy^2(t)}{dt^2} + T_1 \frac{dy(t)}{dt} + y(t) = A \cdot x(t)$$

$$T^2 \frac{dy^2(t)}{dt^2} + 2\xi T \frac{dy(t)}{dt} + y(t) = A \cdot x(t)$$

Transfer functions:

$$G(s) = \frac{A}{T_2^2 s^2 + T_1 s + 1}$$

$$G(s) = \frac{A}{T^2 s^2 + 2\xi T s + 1}$$

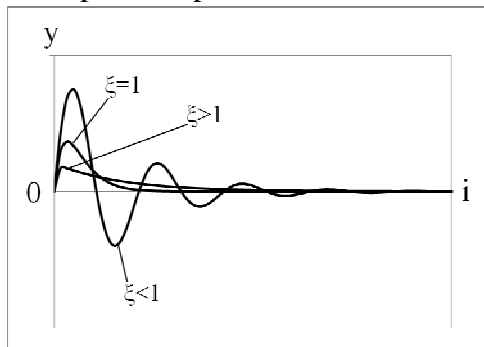
	Impulse responses	Step responses
$\xi < 1$	$\hat{y} = a \cdot A \cdot \left[ \frac{1}{\omega T^2} e^{\alpha t} \sin(\omega t) \right]$ <p>where <math>\alpha = \frac{\xi}{T}</math> and <math>\omega = \frac{1}{T} \sqrt{1 - \xi^2}</math></p>	$\hat{y} = a \cdot A \cdot \left[ 1 - e^{\alpha t} \left( \cos(\omega t) + \frac{\alpha}{\omega} \sin(\omega t) \right) \right]$ <p>where <math>\alpha = \frac{\xi}{T}</math> and <math>\omega = \frac{1}{T} \sqrt{1 - \xi^2}</math></p>
$\xi = 1$	$\hat{y} = a \cdot A \cdot \left[ \frac{1}{T^2} \cdot t \cdot e^{-\frac{t}{T}} \right]$	$\hat{y} = a \cdot A \cdot \left[ 1 - e^{-\frac{t}{T}} \left( 1 + \frac{t}{T} \right) \right]$
$\xi > 1$	$\hat{y} = a \cdot A \cdot \frac{1}{T_1 - T_2} \left[ e^{-\frac{t}{T_1}} - e^{-\frac{t}{T_2}} \right]$	$\hat{y} = a \cdot A \cdot \left[ 1 - \frac{1}{T_1 - T_2} \left( T_1 \cdot e^{-\frac{t}{T_1}} - T_2 \cdot e^{-\frac{t}{T_2}} \right) \right]$

$\xi < 1$  underdamped,       $\xi = 1$  critically damped,       $\xi > 1$  overdamped

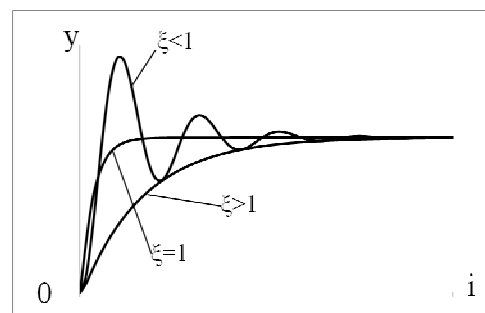
Two basic classes:

1. intrinsic second order elements
2. elements obtained by consecutively connecting two first order elements
  - with equal time constants:  $\xi = 1$
  - with unequal time constants:  $\xi > 1$

Its impulse response:



Its step response:



## VALVES

### Maximum throughput at given pressure drop on the valve

definition of  $k_{v,max}$ :  $W_{max} = k_{v,max} \sqrt{\frac{\Delta p_{rel}}{\rho_{rel}}}$

where  $k_{v,max}$  – water flow rate through perfectly open valve for 1 bar pressure drop over it.

definitions:  $\Delta p_{rel} = \frac{\Delta p_{valve}}{\Delta p_{atm}} = \frac{\Delta p_{valve}}{1\text{bar}}$        $\rho_{rel} = \frac{\rho}{\rho_{water,20^\circ\text{C}}} = \frac{\rho}{1000 \frac{\text{kg}}{\text{m}^3}}$

$W_{max}$  is not constant but depends on pressure drop and density!

### Throughput characteristics:

Linear:  $\frac{W}{W_{max}} = \frac{H}{H_{max}} = h$       (this is definition of h)

Square root:  $\frac{W}{W_{max}} = \sqrt{\frac{H}{H_{max}}} = \sqrt{h}$

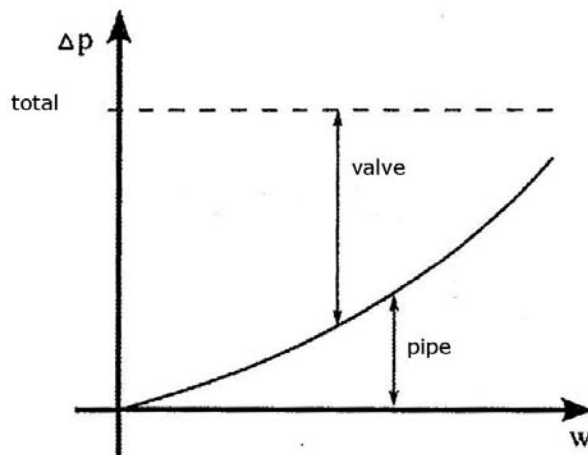
Exponential:  $\frac{W}{W_{max}} = \frac{1}{e^n} \cdot e^{n \cdot \frac{H}{H_{max}}} = \frac{1}{e^n} \cdot e^{n \cdot h}$       (n=1..3)

Non-linearity of the process can be offset with properly selected valves.

### Valve in pipeline:

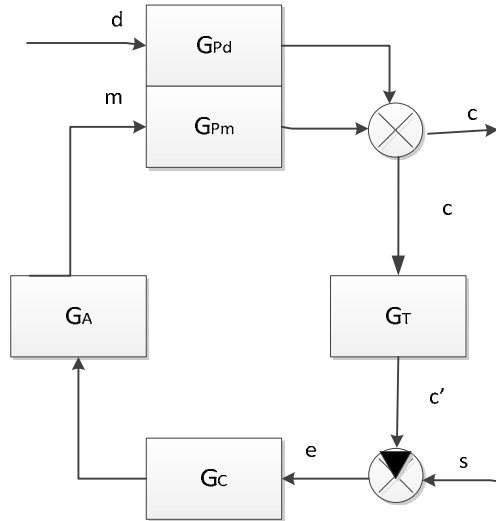
Total pressure drop:  $\Delta p_{total} = \Delta p_{pipe} + \Delta p_{valve}$

Due to friction:  $\Delta p_{valve} = B \cdot W^2$       (assume turbulent flow)

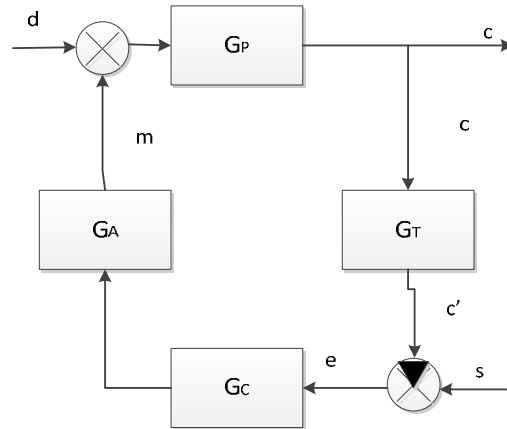


## CONTROL LOOPS

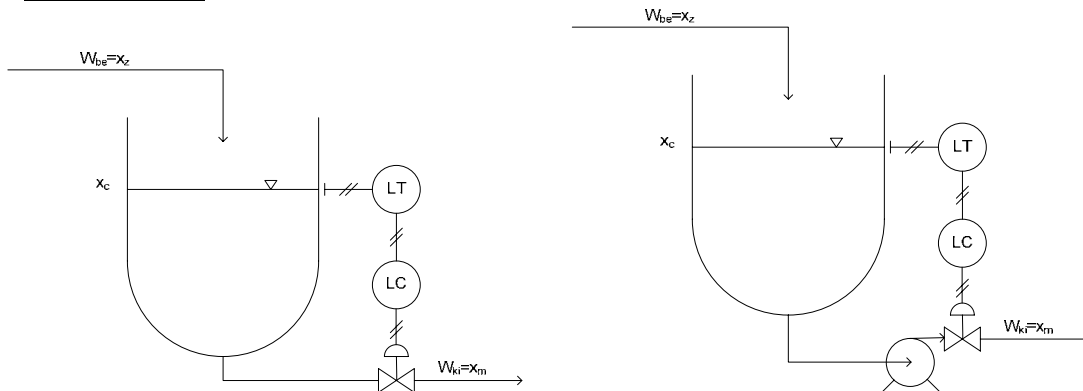
### Block scheme



If the process reacts to  $d$  as  $m$ , then a simplified block scheme can be used:



### Level control



Outflow flow rate is manipulated variable, and inflow flow rate is disturbance.

Process:

Free outflow tank

first order element

$$G_p(s) = \frac{h(s)}{W_{in}(s)} = \frac{A}{T \cdot s + 1}$$

$h$  denotes level (height),  $B$  denotes cross section area

$$A = \frac{2 \cdot h}{W_{in}} \quad T = \frac{2 \cdot B \cdot h}{W_{in}}$$

Forced outflow tank

integrating element

$$G_p(s) = \frac{h(s)}{W_{in}(s)} = \frac{A}{s}$$

$$A = \frac{1}{B}$$

Transmitter – Proportional element

Controller – P controller (proportional element)

Actuator – Valve with linear working characteristic, proportional element

## Closed loop transfer functions:

### Forced outflow tank

Input flow rate  $\Rightarrow$  liquid level:

$$G^*(s) = \frac{h(s)}{W_{in}(s)} = \frac{c(s)}{d(s)} = \frac{G_P(s)}{1 + G_P(s) \cdot G_{Tr}(s) \cdot G_C(s) \cdot G_{Ac}(s)} = \frac{\frac{A_P}{s}}{1 + \frac{A_P}{s} \cdot A_{Tr} \cdot A_C \cdot A_{Ac}}$$

$$G^*(s) = \frac{A_P}{s + A_P \cdot A_{Tr} \cdot A_C \cdot A_{Ac}} = \frac{\frac{A_P}{A_P \cdot A_{Tr} \cdot A_C \cdot A_{Ac}}}{\frac{1}{A_P \cdot A_{Tr} \cdot A_C \cdot A_{Ac}} \cdot s + 1} = \frac{\frac{1}{A_{Tr} \cdot A_C \cdot A_{Ac}}}{\frac{1}{A_P \cdot A_{Tr} \cdot A_C \cdot A_{Ac}} \cdot s + 1} = \frac{A^*}{T^* \cdot s + 1}$$

### Free outflow tank

Input flow rate  $\Rightarrow$  liquid level:

$$G^*(s) = \frac{h(s)}{W_{in}(s)} = \frac{c(s)}{d(s)} = \frac{G_P(s)}{1 + G_P(s) \cdot G_{Tr}(s) \cdot G_C(s) \cdot G_{Ac}(s)} = \frac{\frac{A_P}{T_P \cdot s + 1}}{1 + \frac{A_P}{T_P \cdot s + 1} \cdot A_{Tr} \cdot A_C \cdot A_{Ac}}$$

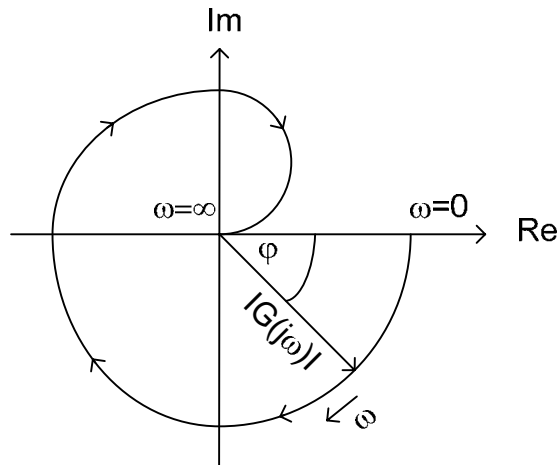
$$G^*(s) = \frac{A_P}{T_P \cdot s + 1 + A_P \cdot A_{Tr} \cdot A_C \cdot A_{Ac}} = \frac{\frac{A_P}{1 + A_P \cdot A_{Tr} \cdot A_C \cdot A_{Ac}}}{\frac{T_P}{1 + A_P \cdot A_{Tr} \cdot A_C \cdot A_{Ac}} \cdot s + 1} = \frac{A^*}{T^* \cdot s + 1}$$

## FREQUENCY ANALYSIS

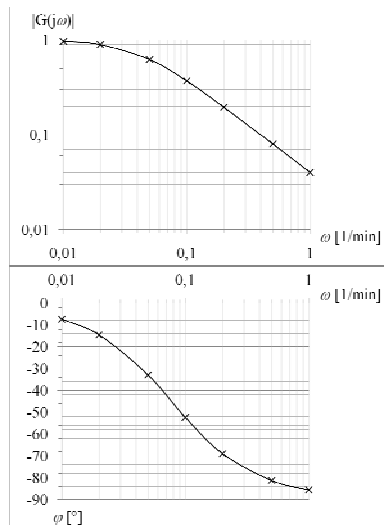
### Frequency function from transfer function

$$G(s) \rightarrow G(j\omega)$$

### Nyquist diagram



### Bode diagram



### One point of the frequency function

$$G(j\omega) = \langle \text{Re} \rangle + j \cdot \langle \text{Im} \rangle$$

$$|G(j\omega)| = \sqrt{\langle \text{Re} \rangle^2 + \langle \text{Im} \rangle^2}$$

$$\varphi = \arctan \frac{\langle \text{Im} \rangle}{\langle \text{Re} \rangle}$$

### Consecutively connected elements:

$$G^*(s) = \prod G_i(s)$$

$$G^*(j\omega) = \prod G_i(j\omega)$$

$$|G^*(j\omega)| = \prod |G_i(j\omega)|$$

$$\varphi^* = \sum \varphi_i$$



### Frequency functions of basic elements

	$G(s)$	$ G(i \cdot \omega) $	$\varphi$
1st order	$\frac{A}{Ts+1}$	$\frac{ A }{\sqrt{1+\omega^2 T^2}}$	$\arctan(-\omega T)$
2nd order	$\frac{A}{T^2 s^2 + 2\xi Ts + 1}$	$\frac{ A }{\sqrt{(1-\omega^2 T^2)^2 + (2\xi T \omega)^2}}$	$\arctan\left(\frac{2\xi \omega T}{1-\omega^2 T^2}\right)$
Dead time	$A \cdot e^{-T_D \cdot s}$	A	$-\omega \cdot T_D$
I	$\frac{A_I}{s} = \frac{1}{I \cdot s}$	$\frac{A_I}{\omega} = \frac{1}{I \cdot \omega}$	$-90^\circ$
PI	$A_C \cdot \left(1 + \frac{1}{I \cdot s}\right)$	$A_C \cdot \sqrt{1 + \frac{1}{I^2 \cdot \omega^2}}$	
PID	$A_C \cdot \left(1 + \frac{1}{I \cdot s} + D \cdot s\right)$		

### Terms

Controlled section: Everything in the loop without the controller itself

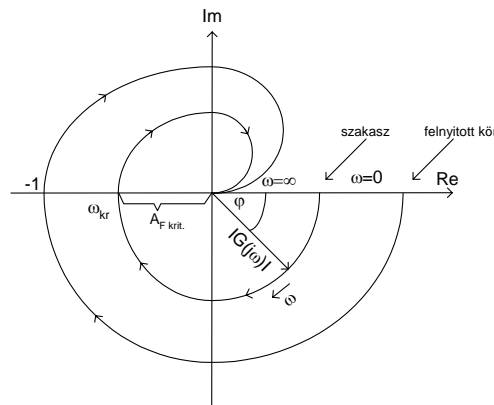
Open loop: Everything in the loop including the controller too

### Critical values

Frequency function of the controlled section  $\rightarrow$  1/critical gain of the section at  $-180^\circ$ .

Frequency function of the open loop with P controller and critical gain intersects  $-1$ .

Frequency function reaches  $-180^\circ$  at critical frequency.



### Tuning

#### Ziegler-Nichols' suggested tuning

based on closed loop cycling or open loop transfer function

	$A_C$	I	D
P	$A_C \leq 0.5 \cdot A_{C,crit}$	$\infty$	0
PI	$A_C \leq 0.45 \cdot A_{C,crit}$	$I \geq 0.8 \cdot T_{crit}$	0
PID	$A_C \leq 0.6 \cdot A_{C,crit}$	$I \geq 0.5 \cdot T_{crit}$	$D < 0.125 \cdot T_{crit}$

### Margins

Phase margin: phase lag of the open loop at amplitude ratio 1 is how much less than (the critical)  $180^\circ$ .

Gain margin: amplitude ratio of the open loop at (the critical) phase lag  $180^\circ$ .